

Vehicle Stability Improvement Based on MFC Independently Installed on 4 Wheels -Basic Experiments using "UOT Electric March II"-

Takahiro Okano, Tai Chien Hwa, Tomoko Inoue,
Toshiyuki Uchida, Shin-ichiro Sakai* and Yoichi Hori
School of Engineering, Department of Electrical Engineering
University of Tokyo
7-3-1 Hongo, Bunkyo-ku, Tokyo
113-8656 Japan
Phone: +81-3-5841-6778, Fax: +81-3-5841-8573
E-Mail: okano@hori.t.u-tokyo.ac.jp

The Institute of Space and Astronautical Science*

Abstract

The focus of our research is on exploiting the excellent control characteristics of the electric motor in realizing advanced vehicle motion control. To this end, we have built an Electric Vehicle (EV) fitted with 4 in-wheel motors. Rapid independent torque control of each motor is realised through a real-time Operating System. With this vehicle we performed experiments to verify control methods which we have formulated, e.g., Model Following Control (MFC). The detection of road conditions and wheel-skid will be planned, too. All these new techniques are possible only on the EV due to its rapid and accurate torque response.

Key words: Electric Vehicle, Motion Control, Anti-lock Braking System, Direct Yaw Moment Control.

1 Introduction

Recently, much research on automobiles with next generation power-trains has been carried out in automobile industries. The improvement of electric vehicles (EVs) has been amazing, and nowadays we see a lot of next generation cars like Prius(TOYOTA) and Insight(HONDA) on the road. The focus of EV research has mainly been on energy and environmental problem. But we overlook other advantages of EVs. These advantages can be summarized as follows,;

1. Electric motor can generate bi-directional torque (accelerating and decelerating) very quickly and accurately.
This is the essential advantage. The electric

motor's torque response is 10-100 times as fast as that of the combustion engine and hydraulic braking system. If we can utilize the fast torque response of the electric motor, applications like "Super TCS"(function as both ABS and TCS) is possible [1].

2. Motor torque can be measured easily.

The torque generation process of the combustion engine and hydraulic brake contains many uncertainties, so it is difficult to accurately measure their output torque. But the electric motor's output torque can be measured easily. Therefore, we can construct a "driving force observer" which observes driving/braking force between the tire and road surface in real-time [2][3]. This advantage will contribute a great deal to several applications like road condition estimation.

3. More than one electric motor can be mounted on each EV.

Electric motors like in-wheel motors are very small. Therefore a motor can be attached to each wheel. In conventional automobile control, Vehicle Stability Control(VSC) like Direct Yaw moment Control(DYC) is very complicated [4][5]. But in EVs, by mounting two or four in-wheel motors, realization of DYC is much easier and its quality is more superior.

In conclusion, these advantages of electric motor give rise to the possibility of vehicle motion control in EVs. In automobile industries, active vehicle control is presently the principal theme. Electrical engineering can contribute much to this novel and important theme. Fig.1 shows the basic idea behind our novel proposal :an integrated system with "minor

These are feedback controllers. Feedback control changes the mechanical system. In this section, we discuss the Model Following Controller "MFC".

3.1 Linear Slip Model

Generally, slip ratio λ is given by,

$$\lambda = \frac{V_w - V}{\max(V_w, V)} \quad (1)$$

Where V is the vehicle chassis velocity, and V_w is the wheel velocity. $V_w = r\omega$, where r, ω are the wheel radius and rotational velocity, respectively.

Motion equations of one wheel model (Fig.7) can be represented as,

$$M_w \frac{dV_w}{dt} = F_m - F_d(\lambda) \quad (2)$$

$$M \frac{dV}{dt} = F_d(\lambda) \quad (3)$$

In these equations, air resistance and rotating resistance are ignored. M is the vehicle weight, M_w is the mass equivalent value of the wheel inertia, F_m is the force equivalent value of accelerating/decelerating torque, and F_d is the driving/braking force between the wheel and the road surface. F_d is a function of λ (Slip Ratio) as is shown in Fig.8.

In order to design the anti-slip controller, nonlinear property in the $\mu - \lambda$ curve should be linearized. We consider small variation around the operational point.

$$dF_d = Nd\mu = aNd\lambda \quad (4)$$

$$= -\frac{1}{V_{w0}}dV + \frac{V_0}{V_{w0}^2}dV_w \quad (5)$$

V_{w0} and V_0 are the wheel velocity and vehicle velocity at the operational point respectively. a is the gradient of $\mu - \lambda$ curve described as

$$a = \frac{d\mu}{d\lambda} \quad (6)$$

Using (1)–(6), we obtain the transfer function from F_m to V_w as follows.

$$P(s) = \frac{dV_w}{dF_m} = \frac{1}{(M_w + M(1 - \lambda_0))s} \frac{\tau_\omega s + 1}{\tau_a s + 1} \quad (7)$$

$$\tau_a = \frac{M_w V_{w0}}{aN} \frac{M}{M(1 - \lambda_0) + M_w} \quad (8)$$

$$\tau_\omega = \frac{MV_{w0}}{aN} \quad (9)$$

In these equations, λ_0 is the slip ratio at the operational point.

Finally, we obtain the simplest transfer functions.

$$P_{adh} = \frac{1}{M + M_w} \frac{1}{s} \quad (10)$$

$$P_{skid} = \frac{1}{M_w} \frac{1}{s} \quad (11)$$

In the next section, we will discuss how to design the Model Following Controller "MFC".

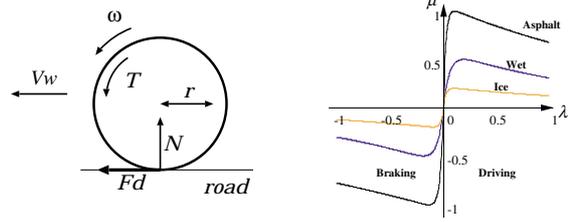


Fig. 7. One Wheel Model Fig. 8. Typical $\mu - \lambda$ Curve

3.2 Controller Design

In this section, we design the Model Following Controller. When the a vehicle starts skidding, the wheel velocity changes rapidly. For example, if vehicle starts skidding during acceleration, its wheel velocity increases rapidly, and during deceleration, it decreases rapidly due to the wheel lock. According to equation (11) the rapid change of wheel velocity is observed as a sudden drop of wheel inertia. Based on this point view, we design the feedback controller "Model Following Controller" as in Fig.9. Using (10) as the nominal model, this controller can suppress sudden drop of inertia. Applying this controller, the dynamics of the skidding wheel becomes close to that of the adhesive wheel. In other words, the wheel to which the proposed controller is applied becomes insensitive to the slip phenomenon.

In the following section, we apply our proposed controller to "UOT Electric March II".

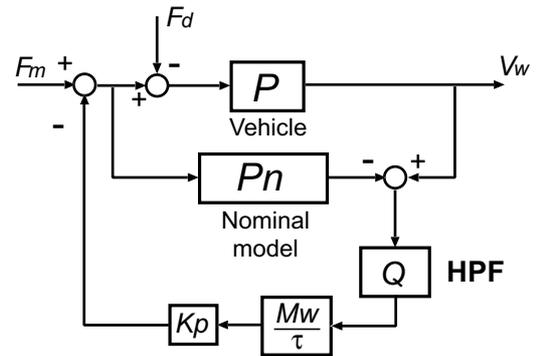


Fig. 9. Block diagram of the proposed feedback controller "MFC"

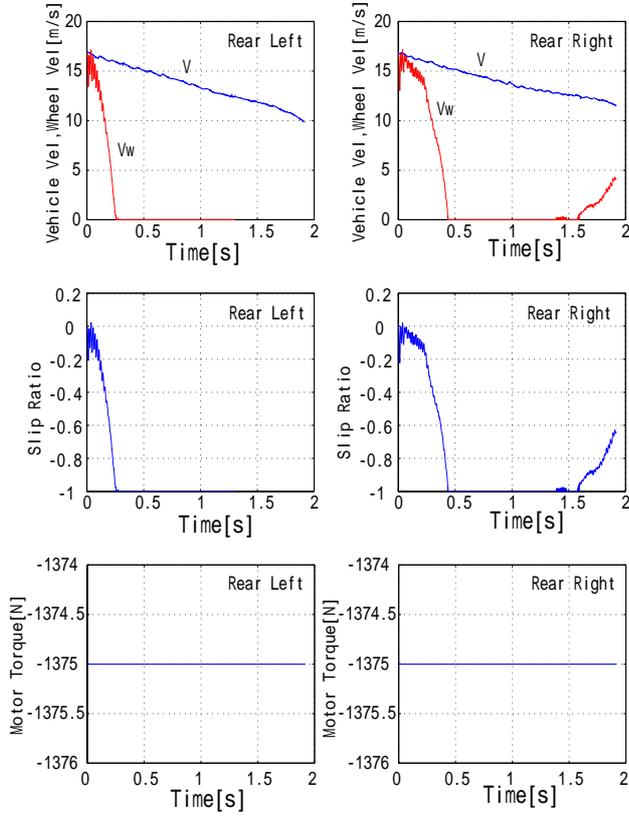


Fig. 10. Wheel lock in rapid braking "without MFC"

4 Experimental Results of MFC with "UOT Electric March II"



Fig. 12. Braking Experiment of "UOT Electric March II"

4.1 Improvement in Braking Performance with MFC

In this section, we discuss the experimental results. In the first experiment, sudden brake is applied on slippery low μ road (Fig.12). μ_{peak} of the experimental road is about 0.5.

Figs.10 and 11 show the experimental results. In these experiments, "UOT Electric March II" decelerated suddenly on the slippery test course. Without control, the wheel velocity rapidly decreased and the vehicle's wheels were soon locked (Fig.10). On the contrary, the change in wheel velocity

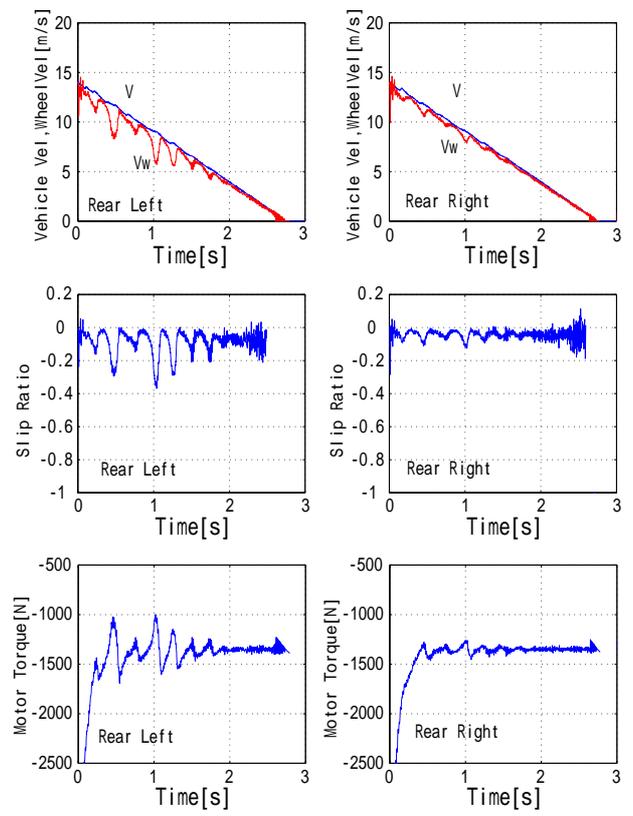


Fig. 11. Stable braking with our proposed controller "MFC"

is relatively slow when the proposed method is applied(Fig.11). The vehicle's wheels did not lock, and the vehicle stopped safely. In this case, the wheel equivalent inertia during the wheel skidding became "heavy" by the effect of MFC, and rapid increase of the slip ratio could be suppressed, and wheel lock were finally avoided.

4.2 Vehicle Stability Improvement with MFC

In the previous section, we discussed the wheel velocity feedback method "MFC". This method suppresses the rapid change in slip ratio and wheel velocity. In this section we will discuss what happens if we apply MFC to each wheels when the vehicle is turning on slippery road. It is common for vehicle's lateral motion to fall into an unstable state, when sudden braking or turning is commanded on slippery road. In these experiments, "UOT Electric March II" did a turn on a slippery road, known as the skid pad. The rear-wheel velocities are controlled independently by the 2 rear motors. ("UOT Electric March II" has one motor mounted on each wheel.)

At first "UOT Electric March II" was turning normally in the clock wise direction. The turning radius is about 25-30[m] and chassis velocity is about

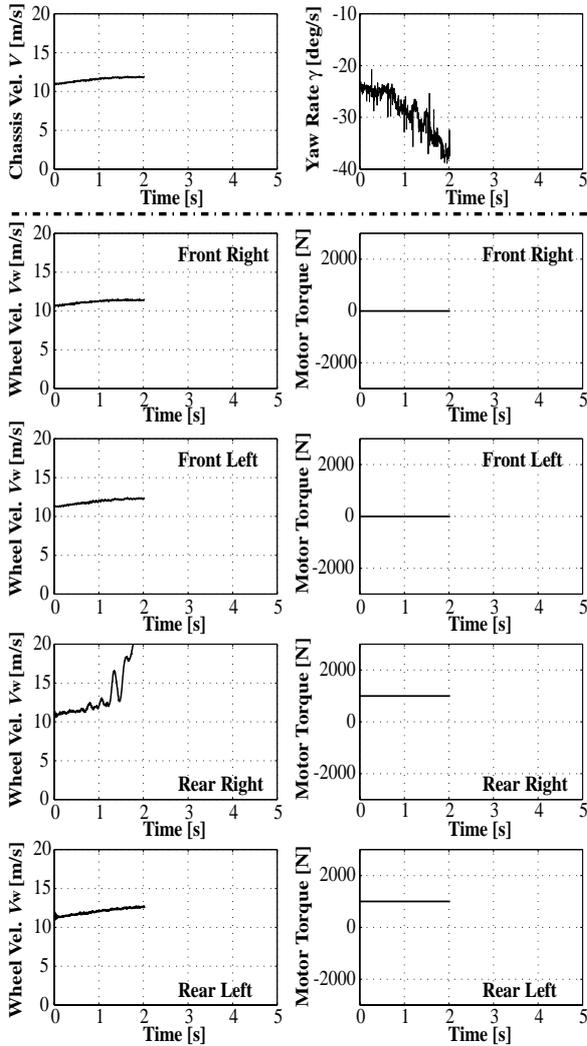


Fig. 13. Unstable turning with sudden acceleration "without MFC"

40[km/h]. These values are close to the unstable region. In these experiments, acceleration torque of 1000[N] was applied to the 2 rear motors. Without MFC, this rapid acceleration torque causes instability (Fig.13). The rear right wheel began skidding dangerously. Then the yaw rate γ grew unstable as shown in Fig.13. This vehicle was in spin motion and completely out of control.

On the contrary, such dangerous vehicle motions could be prevented with our proposed method "MFC". Figs.14 and 15 show this effect clearly. And Fig.16 is a comparison of the vehicle's trajectories. It shows that the MFC controller prevents spin out due to excessive over steer.

In this case the controllers on rear-right and rear-left are the same but independent from each other, yet vehicle stability is preserved. In other words, autonomous stabilization of each driven wheel was achieved, and vehicle lateral stability was enhanced, as is observed in DYC.

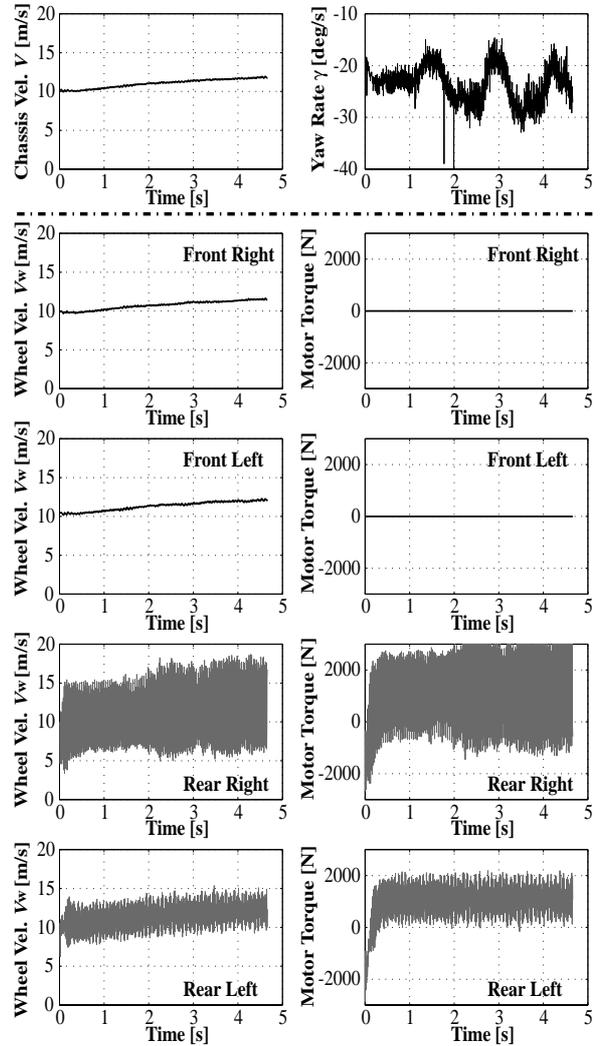


Fig. 14. Vehicle stabilizing effect of our proposed controller "MFC"

One of the remaining problems is the high-frequency oscillation of the rear wheels. It appears in Figs.14 and 15. It is probably due to the design of the controller's parameters. We will solve this problem in our next experiment.

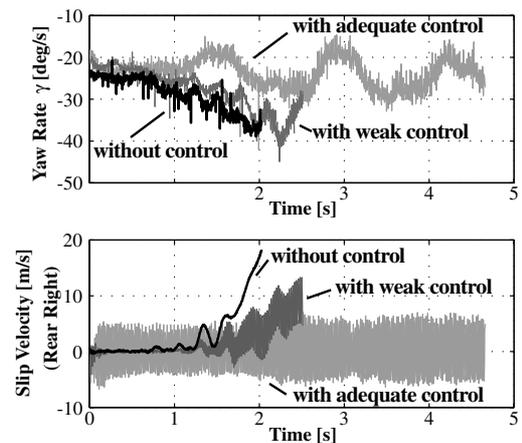


Fig. 15. Comparison of Vehicle Value " γ, V_w "

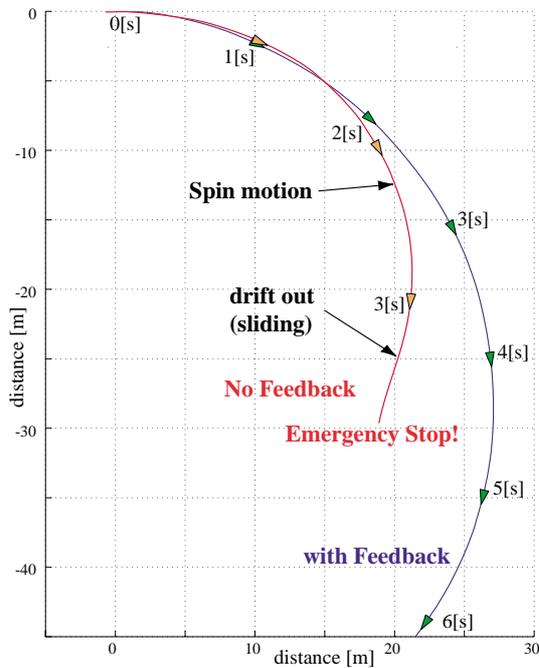


Fig. 16. Stabilizing Effect of “MFC Controller”

5 Conclusion

In this paper, we introduced our novel experimental EV “UOT Electric March II”. This new 4 motored EV will play an important role in our novel motion control studies. As the first attempt, we proved the effectiveness of “MFC” using the vehicle.

The most remarkable point of our research is in utilization of the electric motor’s advantage: quick, accurate and distributed torque generation. Recent concerns on EV is mainly on energy and environment, but we believe that, in future, high performance vehicle stability control will be the major topic, which can be firstly realized by EV’s.

6 Future Research

In this paper, we discussed “MFC”, but we have studied on several other motion control issues revolving around EVs. For example, “Road Condition Estimation”[2], “Vehicle Velocity Estimation”, “ β (Chassis Slip Angle) Estimation” “Decoupling of Direct Yaw Moment Control and Active Front Steering” and “Hybrid ABS” [6].

In the near future, we will carry out experiments on these topics using “UOT Electric March II”.

References

- [1] Y. Hori, Y. Toyoda and Y. Tsuruoka, “Traction control of electric vehicle: Basic experimental results using the test EV “UOT electric march””, IEEE Trans. Ind. Applicat., vol.34, No.5, pp.1131-1138, 1998.
- [2] Hideo Sado, Shin-ichiro Sakai and Yoichi Hori, “Road condition estimation for traction control in electric vehicle”, in The 1999 IEEE International Symposium on Industrial Electronics, pp.973-978, Bled, Slovenia, 1999.
- [3] Shin-ichiro Sakai, Hideo Sado and Yoichi. Hori, “Novel wheel skid detection method for electric vehicles”, in Proc. The 16th. Electric Vehicle Symposium (EVS16), pp.75, Beijing, China, 1999.
- [4] Yasuji Shibahata et al., “The improvement of vehicle maneuverability by direct yaw moment control”, in Proc. 1st International Symposium on Advanced Vehicle Control, No.923081, 1992.
- [5] Sumio Motoyama et al., “Effect of traction force distribution control on vehicle dynamics”, in Proc. 1st International Symposium on Advanced Vehicle Control, No.923080, 1992.
- [6] Shin-ichiro Sakai and Yoichi. Hori, “Advanced vehicle motion control of electric vehicle based on the fast motor torque response”, in Proc. 5th International Symposium on Advanced Vehicle Control, pp.729-736, Michigan, USA, 2000.
- [7] Y. Furukawa and M. Abe, “Direct yaw moment control with estimating side-slip angle by using on-board-tire-model”, in Proc. 4th International Symposium on Advanced Vehicle Control, pp.431-436, Nagoya, 1998.